CFD simulations – some information about scientific computing.

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- What is CFD simulation?
- Why should we use CFD simulations.
- CFD simulation important aspects.
- Start with simple convection diffusion example:
 - Finite volume discretization and solution of discrete system.
 - Simulation examples.



Computational Fluid Dynamics or CFD is the analysis of systems involving fluid flow, heat transfer and associated phenomena such as chemical reactions by means of computer-based simulation. The technique is very powerful and spans a wide range of industrial and non-industrial application areas (*).

Some examples are:

- aerodynamics of aircraft and vehicles: lift and drag
- turbomachinery: flows inside rotating passages, diffusers etc.
- electrical and electronic engineering: cooling of equipment including micro- circuits
- chemical process engineering: mixing and separation, polymer molding, foam expansion flows.
- external and internal environment of buildings: wind loading and heating/ ventilation
- meteorology: weather prediction
- biomedical engineering: blood flows through arteries and veins

(*) An introduction to computational fluid dynamics. The finite volume method. H.K.Versteeg, W.Malalasekera.



- Computers are becoming more and more powerful.
- CFD simulations allows us to:
 - Fast prediction of flow behavior (depending on problem formulation).
 - Help the engineers to develop new products
 - The simulation enables the inspection of product parts that are not visible in the test setup.
 - Simulation enables engineers to make faster and better decisions.
 - Simulation improves product quality, durability, safety, and performance.
 - Simulation may reduce development time and costs significantly.
 - This speed leads to increased productivity and more efficient use of engineering resources.
- Don't forget: CFD simulation is fun ☺



• CFD is by its nature complex because it combines several components, each of which is a challenge in its own right:

- fluid dynamics and physical modelling;
- geometry and meshing;
- numerical methods discretization of equations and solution of discrete system;
- data analysis;
- computing and programming
- It is therefore difficult to achieve CFD competence, i.e. to have the confidence to perform CFD analyses repeatedly and on a timely basis according to a defined standard.
- Many people often underestimate the complexity of CFD and assume that the required competence can be reached simply by "learning the software package".



CFD work layout:

- Define your problem;
- STEP 1: create mathematical model describing your process.
- STEP 2: discretize your equations on a given mesh.
- STEP 3: solve discrete equation (or system of equations).
- STEP 4: visualize and analyze simulation results.
- For STEP 1 STEP 3 we introduce errors.
 - STEP 1: assumptions in model description.
 - STEP 2: errors in discretization.
 - STEP 3: errors when solving discrete equation.
- Goal: try to minimize errors in all 3 steps.



Convection – diffusion equation.

Equation and meaning.

■ We define following convection – diffusion equation:

$$\frac{\partial(\rho\phi)}{\partial t} + div(\rho\phi\mathbf{u}) = div(\Gamma \ grad \ \phi) + S_{\phi}$$

• Where ϕ is variable to solve, ρ – density, Γ – diffusion coefficient, u – velocity, S_{ϕ} - source term.

Meaning of the terms:

Rate of increase	Net rate of flow	Rate of increase	Rate of increase
of ϕ of fluid	+ of ϕ out of =	= of ϕ due to +	of ϕ due to
element	fluid element	diffusion	sources



There are various methods for discretizing the governing equations called discretization methods.

- Some popular discretization methods used in CFD tools are
 - the finite element method,
 - the finite volume method
 - the finite difference method
- For CFD simulations most popular is Finite Volume Method (FVD).
- Advantages of FVM:
 - The FVM is a natural choice for solving CFD problems because the PDEs you need to solve for CFD are conservation laws
 - The biggest advantage of the FVM is that it only must perform a flow evaluation for the cell boundaries.
 - FVM conserves the quantities.



• We integrate equation over control volume (CV). CV can be associated to single mesh element.

$$\int_{CV} \frac{\partial(\rho\phi)}{\partial t} \, dV + \int_{CV} div(\rho\phi\mathbf{u}) dV = \int_{CV} div(\Gamma \text{ grad } \phi) dV + \int_{CV} S_{\phi} dV$$

Now we make use of Gauss divergence theorem

$$\int_{CV} div \, \mathbf{a} dV = \int_{A} \mathbf{n} \cdot \mathbf{a} dA$$

• where A denotes surface of control volume and n – normal to surface vector.



Using Gauss divergence theorem we convert our equation to:

$$\frac{\partial}{\partial t} \left(\int_{CV} \rho \phi dV \right) + \int_{A} \mathbf{n} \cdot (\rho \phi \mathbf{u}) dA = \int_{A} \mathbf{n} \cdot (\Gamma \ grad \ \phi) dA + \int_{CV} S_{\phi} dV$$

terms meaning:

Rate of increase of ϕ	Net rate of $+ \frac{\text{decrease of } \phi \text{ due to}}{\text{convection across}} = \frac{1}{2}$	Rate of increase of ϕ due to diffusion across the boundaries	+ Net rate of creation of ϕ
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In discrete form we write equation volumetric terms as:

Time derivative term:

$$\frac{\partial}{\partial t} \left(\int_{CV} \rho \phi \, dV \right) = \frac{\partial(\rho \phi)}{\partial t} \, \delta CV$$

Source term:

$$\left(\int\limits_{CV} S_{\phi} \, dV\right) = S_{\phi} \,\delta CV$$



Control volume (CV)



Convection – diffusion equation.

Space discretization.

In discrete form we write equation surface terms as:

Convection term:

$$\int_{A} n(\rho \phi u) \, dA = \sum_{f = \{e, w, n, s, t, b\}} n_f A_f \rho_f \phi_f u_f$$

Diffusion term:

$$\int_{A} n (\Gamma \nabla \phi) dA = \sum_{f \in \{e, w, n, s, t, b\}} n_f A_f \Gamma_f \nabla \phi_f$$



Control volume (CV)



Convection – diffusion equation.

Space discretization.

• We restrict discretization to x-axis (faces e,w) and assume constant density ρ :

- Convection term, many different schemes could be used.
- Simple 1st order accuracy in space $O(\delta x)$ scheme UPWIND reads as:

$$A_e \rho_e \phi_e u_e = \begin{cases} \rho A_e \phi_P u_e & if u_e \ge 0\\ \rho A_e \phi_E u_e & if u_e < 0 \end{cases}$$

Diffusion scheme:

$$A_e \Gamma_e \nabla \phi_e = A_e \Gamma_e \frac{\phi_E - \phi_P}{\delta x} \qquad \Gamma_e = \frac{\Gamma_E + \Gamma_P}{2}$$

Gradient approximated by truncated Taylor expansion (second order space accuracy $O(\delta x^2)$). Arithmetic average for diffusion coefficient could be used.





Let assume time space is in steps with time step dt, we denote "new" time step values with superscript "t + 1" and previous time step values with superscript "t"

Time derivative term is now approximated with backward Euler 1st order time approximation $O(\delta t)$:

$$\frac{\partial(\rho\phi)}{\partial t}\delta CV = \rho \frac{\phi_P^{t+1} - \phi_P^t}{dt}\delta CV$$

Final decision is which scheme to use:

- Explicit all discretized terms used from previous time step, all ϕ values used as ϕ_P^t , ϕ_E^t
- Implicit all discretized term used in new time step, all ϕ values used as ϕ_P^{t+1} , ϕ_E^{t+1}

Explicit scheme straightforward to implement, however such schemes require time step stability restrictions:

Convection: The Courant–Friedrichs–Lewy (CFL) condition: $CFL = \frac{dt \cdot u}{dx} < 1$

Diffusion: $dt < \frac{1}{2} \frac{dx^2}{\Gamma}$



- Fully implicit time discretization is unconditionally stable.
- However, it requires solution of linear algebra systems.
- We convert discretized system into algebraic system of the form: $\overline{A} \cdot x = b$
- Coefficients appearing in all discretized term build up matrix \bar{A}
- Source term, time step part (with ϕ_P^t) and discretized terms from outer geometry boundary parts build up vector b
- Vector $x = \{\dots, \phi_W^{t+1}, \phi_P^{t+1}, \phi_E^{t+1}, \dots\}$
- Many different methods to solve such systems, like iterative methods (CG, GMRES, BiCGstab, multigrid)
 To speedup solution of linear algebra systems preconditioners used. Here one tries to approximate M ≈ A
 , where matrix M is easily invertible. Then one solves M⁻¹A
 · x = M⁻¹b. Note that M⁻¹A
 has lower condition number, which results in faster convergence.



Convection – diffusion equation. Implicit vs explicit discretization summary.

Fully implicit time discretization:

$$\rho \frac{\phi_P^{t+1} - \phi_P^t}{dt} \delta CV + \sum_{f = \{e, w, n, s, t, b\}} n_f A_f \rho_f \phi_f^{t+1} u_f = \sum_{f = \{e, w, n, s, t, b\}} n_f A_f \Gamma_f \nabla \phi_f^{t+1} + S_\phi \delta CV$$

• Fully explicit discretization:

$$\rho \frac{\phi_P^{t+1} - \phi_P^t}{dt} \delta CV + \sum_{f = \{e, w, n, s, t, b\}} n_f A_f \rho_f \phi_f^t u_f = \sum_{f = \{e, w, n, s, t, b\}} n_f A_f \Gamma_f \nabla \phi_f^t + S_\phi \delta CV$$

Mixed implicit – explicit discretization possible.



Convection – diffusion equation.

Boundary conditions.

• We will consider 3 different boundary conditions:

Dirichlet boundary: $\phi_{wall} = ar{\phi}$, here we fix value at the boundary "wall" to pre-defined $ar{\phi}$

Neumann boundary: $\frac{\partial \phi_{wall}}{\partial n} = \bar{\phi}$, here we fix normal gradient at the boundary wall to pre-defined $\bar{\phi}$. Note: when $\bar{\phi}=0$, we assume no change of ϕ in the wall direction.

Flux condition: $\Gamma \nabla \phi \cdot n = \alpha_w (\phi_{wall} - \phi_{ext}) \cdot n$, here be balance fluxes from internal geometry part with flux from outside computational part by assuming external value ϕ_{ext} and parameter α_w . Note, for very high values of α_w this boundary condition approaches Dirichlet boundary (with value ϕ_{ext}) and for very low values of α_w we approach zero Neumann boundary condition.



 Forthcoming simulation results are performed on 2D square geometry of size [0,0.2]x[0,0.2] (m)

• We start with simple diffusion equation:

 $\frac{\partial T}{\partial t} = \Gamma \Delta T$

- We will test different:
 - Initial conditions
 - Boundary conditions (E,W,N,S)
 - Diffusivity values.





Case 1:

- Diffusion value Γ=0.0001
- Initial value 22;
- Boundary condition (Dirichlet): $T_W = 40, T_E = 40, T_N = 40, T_S = 40$
- As expected, uniform steady state solution.







Case 2:

- Diffusion value Γ=0.0001
- Initial value 22;

Boundary condition: (Dirichlet) $T_W = 40, T_E = 20$,(Neumann) $\frac{\partial T_N}{\partial n} = 0, \frac{\partial T_S}{\partial n} = 0$

As expected, linear drop from value 40 to 20 in x-axis direction and uniform distribution in y-axis direction.







T=20

0.20

Case 3:

- Diffusion value Γ=0.0001
- Initial value 22;
- Boundary condition: $T_W = 35, T_E = 22, T_N = 40, T_S = 28$.







Case 4:

- Diffusion value Γ=0.00001
- Initial value: non-uniform;
- Boundary condition: $T_W = 40, T_E = 40, T_N = 40, T_S = 40$.
- Uniform steady state solution.









Case 5:

- Diffusion value Γ=0.0001,0.00001, 0.000001
- Initial value: non-uniform;
- Boundary condition: $T_W = 40$, $T_E = 40$, $T_N = 40$, $T_S = 40$.

Diffusion parameter has impact on speed of diffusion.





Case 5:

- Diffusion value Γ=0.0001,0.00001, 0.000001
- Initial value: non-uniform;
- Boundary condition: $T_W = 40$, $T_E = 40$, $T_N = 40$, $T_S = 40$.

Diffusion parameter has impact on speed of diffusion and smooths out initial oscillations.





Case 6:

- Diffusion value Γ=0.0001
- Initial value: 22;
- Boundary condition 1: $T_W = 40, T_E = 40, T_N = 40, T_S = 40$.
- Boundary condition 2: Flux on all boundaries with α_w =0.5 and T_{ext} = 40
- Diffusion parameter has impact on speed of diffusion and smooths out initial oscillations.





Forthcoming simulation results are performed on 2D square geometry of size [0,0.2]x[0,0.2] (m)

We consider simple convection equation:

$$\frac{\partial s}{\partial t} + u \cdot \nabla s = 0$$

- We will test different:
 - Initial conditions
 - Boundary conditions (E,W,N,S)
 - Velocity fields.
 - Different diffusion schemes.

Note, that all convection schemes show artificial diffusion that

may affect accuracy of the solution.





Case 1:

- Constant convection velocity: u = (0.02, 0).
- Initial value: s = 0;
- Boundary condition: $s_W = 1$, $\frac{\partial s_E}{\partial n} = 0$, $\frac{\partial s_N}{\partial n} = 0$, $\frac{\partial s_S}{\partial n} = 0$.
- 4 different convection schemes used.
 - Upwind
 - TVD superbee
 - ADBQUICKEST
 - STACS

Note, we do not discuss in detail implementation issues for all those schemes.





Simulation examples.

Convection equation.





• We consider now Upwind scheme.

Using Taylor expansion one can determine the local truncation error of discretization scheme.

For upwind scheme one can show that dropping first order terms from Taylor expansion

leads to artificial numerical diffusion term: $\frac{u \cdot dx}{2} (1 - CFL) \frac{\partial^2 s}{\partial x^2}$

- Recall, *CFL* is Courant–Friedrichs–Lewy value.
- This means that we can reduce numerical diffusion in 2 ways:
 - use finer mesh (reduction of dx)
 - use time step such that CFL = 1, however for explicit scheme this may lead to

instabilities. For CFL > 1 explicit schemes are instable.

We consider now Upwind scheme for different simulation setup.



• Note that for fluid dynamics application no possible to choose time step to satisfy $CFL \approx 1$ in all grid elements.



- Case 2 so called Zalesak disc:
 - rotational velocity wit center of 0.1.
 - Initial value: disc with cut out located away from middle point (0.1,0.1);
 - Boundary condition: $\frac{\partial s_W}{\partial n} = 0$, $\frac{\partial s_E}{\partial n} = 0$, $\frac{\partial s_N}{\partial n} = 0$, $\frac{\partial s_S}{\partial n} = 0$.
- 4 different convection schemes used.
 - Upwind
 - TVD superbee
 - ADBQUICKEST
 - STACS
- Simulation stops after one full revolution.





- Case 2: Zalesak disc, one revolution
- Simulation time 1 (s).





Case 2:

Simulated time: 1(s)

 Only STACS retains original shape after one revolution.
 However, also for STACS front diffuses as well.







Summary and outlook.

- Short introduction to scientific computing has been presented.
- Several simulation examples of convection, diffusion type have been showed and discussed.
- REMEMBER: Do not trust any simulation tool ! Always analyze simulation results to be more confident that simulation provide correct output.
- Next lecture:
 - Some aspects of flow equations will be discussed.
 - Newtonian, non-Newtonian flows.
 - Industrial application examples.





